

A Dynamic 3-D Surface Profilometer With Nanoscale Measurement Resolution and MHz Bandwidth for MEMS Characterization

Liang-Chia Chen, Yao-Ting Huang, and Kuang-Chao Fan

Abstract—Commercialization of microelectromechanical systems (MEMS) has made accurate dynamic characterization a major challenge in design and fabrication. In view of this need, a dynamic 3-D surface profilometer involving white light interferometric scanning principle with a stroboscopic LED light source was developed. The developed instrument was applied to a microcantilever beam used in atomic force microscopy (AFM) to analyze its full-field resonant vibratory behavior. The first five resonant vibration modes were fully characterized with vertical measurement accuracy of 3–5 nm and vertical measurement in the range of tens of micrometers. The experimental results were consistent with the outcomes of the theoretical simulation by ANSYS. Using stroboscopic illumination and white light vertical scanning techniques, the developed static and dynamic 3-D nanoscale surface profilometry of MEMS devices can achieve measurement range of tens of micrometers and dynamic bandwidth of up to 1-MHz resonance frequency.

Index Terms—Dynamic profilometry, integrated mechatronics, microelectromechanical systems (MEMS) dynamic characterization, stroboscopic interferometry.

I. INTRODUCTION

MICROELECTROMECHANICAL systems (MEMS), such as microaccelerometers, microbeams, micromembranes, and microbridges, possess component or system functionality essentially relying on the dynamic displacement properties of the microstructure and accurate characterization. In particular, this kind of characterization requires comprehensive knowledge of the vibration behavior of the MEMS. The design, performances, and reliability of MEMS and microoptoelectromechanical systems (MOEMS) depend critically on the control of the whole technology and especially on the knowledge and control of the mechanical behavior of materials and micromechanical devices [1], [2]. Characterization of the actual mechanical behavior of MEMS is essentially required since theoretical simulation cannot be performed due to possible dimensional imperfections, unexpected effects from inherent stress

Manuscript received March 1, 2006; revised December 15, 2006. Recommended by Guest Editors H.-P. Huang and F.-T. Cheng. This work was supported by the Center for Measurement Standards (CMS) of the Industrial Technology Research Institute (ITRI), Taiwan, R.O.C., under a Cooperative Research Grant in 2005.

L.-C. Chen is with the Graduate Institute of Automation Technology, National Taipei University of Technology, Taipei 106, Taiwan, R.O.C. (e-mail: lchen@ntut.edu.tw; s3618004@ntut.edu.tw).

K.-C. Fan is with the Department of Mechanical Engineering, National Taiwan University, Taipei 106, Taiwan, R.O.C., and also with Hefei University of Technology, Hefei 3345881, China (e-mail: fan@ccms.ntu.edu.tw).

Digital Object Identifier 10.1109/TMECH.2007.897268

gradient, unpredictable real-boundary conditions, and damping mechanisms. Additionally, microstructure resonant frequencies can range from hertz to several megahertz, which significantly limits the range of measurement due to the increase in measurement bandwidth [3]. When performing stroboscopic interferometric measurement, the golden rule for freezing the moving interferometric fringe is that the larger the vibrating frequency is, the shorter the detecting strobed light should flash. When the vibrating period decreases to less than a few hundred nanoseconds, an excessively lengthy flashing time can lead to the degradation of interferograms caused by unwarranted image blurring.

In general, atomic force microscopy (AFM) possesses an atomically sharp probe tip affixed to a cantilevered beam, which is raster-scanned in close proximity over the surface of interest [4]. Most AFM cantilever beams are made of monocrystalline silicon. The resonance frequency and the force constant are mainly determined by the geometry and material properties of the cantilever beams. The thickness of the cantilever beam is generally measured using an interferometric microscope while the length and width are measured with an optical microscope. The resonance frequency and force constant can be estimated using the conventional flexural vibration equations. However, possible measurement errors and the assumption used in the calculation may cause 10%–20% errors when predicting the resonance frequency and force constant of cantilevers of 125–450 μm long [5]. Furthermore, for microdevices with internal residual stress, the vibration modes may significantly differ from that of unstressed microdevices [6]. Therefore, it is obvious that a single spectrum and point-type characterization are generally not sufficient for evaluating the dynamic behavior of either stressed elementary or complex AMF cantilever beams [4].

Heterodyne laser vibrometers, involving the laser Doppler effect and extensively used for MEMS vibration spectra measurements, have a typical detection limit below 10 pm in a frequency bandwidth of a few megahertz. They are insensitive to environmental noises and good for out-of-plane measurement [2]. However, such technique only obtains a point measurement of the out-of-plane vibrations and does not provide the essential information on the relative phase of the vibrations, thus, affecting the comprehensive understanding of dynamic behaviors of microdevices. To enhance the scanning efficiency, optical microscopic interferometry, digital holography (DH), and electronic speckle pattern interferometry (ESPI) can all be applied to full-field out-of-plane vibration measurements of M(O)EMS using either time averaging or stroboscopic techniques. Time-averaged interferometry with a fringe contrast function can

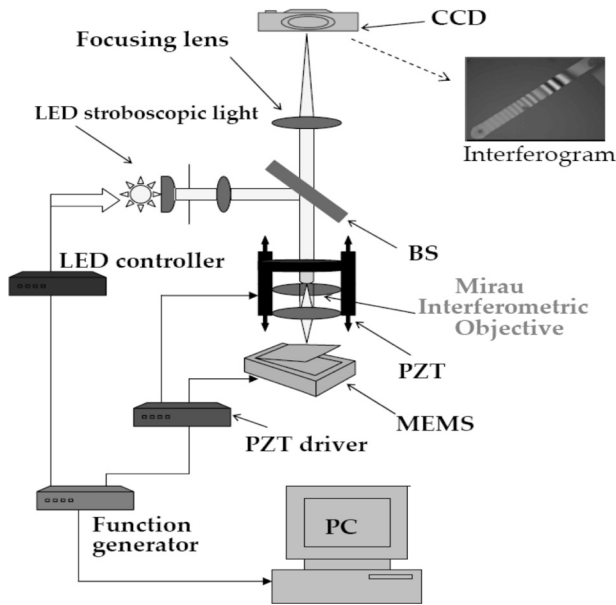


Fig. 1. Schematic diagram of the developed optical system.

87 perform quantitative analysis of the interference pattern in
 88 contrast to obtaining vibration mode shapes, but is only suit-
 89 able for low vibration frequency measurement. To overcome
 90 this problem, time-resolved stroboscopic measurements using
 91 either white light, single-wavelength LED light, or pulsed laser
 92 can be deployed for full-field interferometric techniques for 2-D
 93 or 3-D measurements of vibration mode shapes with a theoret-
 94 ical measurement bandwidth of up to 2 MHz [7]. The strobo-
 95 scopic interferometry can be combined with phase-shifting
 96 algorithm as well as with fast Fourier transform (FFT) interfe-
 97 rogram processing to obtain quasi-real-time 3-D profilometry of
 98 the vibration mode shape. Thus, this study develops a micro-
 99 scopic measurement system, involving white light stroboscopic
 100 interferometry and vertical scanning principle, to achieve full-
 101 field 3-D measurements in the range of tens of micrometers
 102 with nanoscale vertical resolution and high-bandwidth response
 103 of up to 1 MHz.

104 This paper comprises five sections organized as follows.
 105 Section II describes the system layout of the developed opti-
 106 cal measurement system with its design of strobed LED light
 107 source and signal-synchronizing electronics. Stroboscopic vi-
 108 bration measurements analysis and interferogram processing
 109 are detailed in Section III. To demonstrate the feasibility of
 110 the proposed methodology, theoretical evaluation and dynamic
 111 vibratory measurements on AFM microcantilever beams with
 112 various operation modes were performed and analyzed in Sec-
 113 tion IV. Section V contains the conclusion and summarizes the
 114 development.

115 II. SYSTEM DESIGN AND INTEGRATION

116 The optical system is established on a white light Mirau
 117 optical interferometer. As shown in Fig. 1, a standard Nikon
 118 microscope is equipped with Mirau interferometric objectives
 119 and a Physik Instrumente (PI) piezoelectric vertical translation

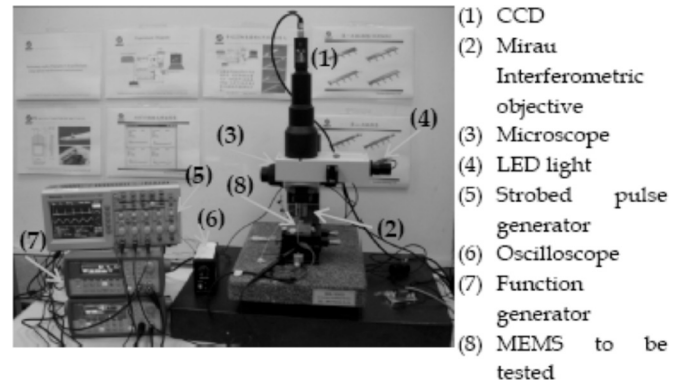
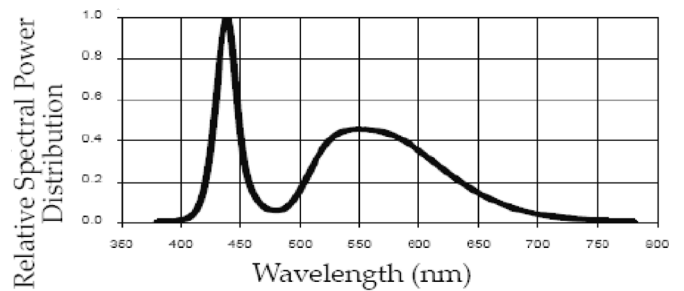
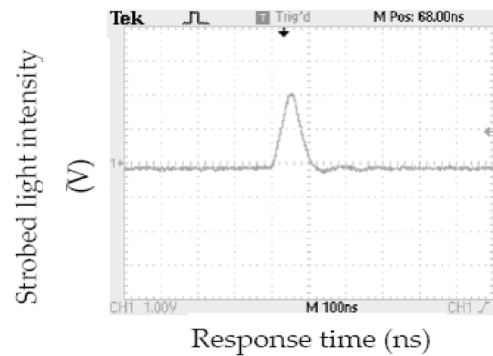


Fig. 2. Hardware setup of the developed optical system.



(a)



(b)

Fig. 3. Property of the LED light source. (a) Light spectrum. (b) Time response of the strobed LED light.

120 system sensed by a capacitive positioning device for with the
 121 closed-loop control. The piezoelectric transducer with embed-
 122 ded capacitive sensors for closed-loop control has a 100- μm
 123 vertical translation range and a subnanometric resolution. The
 124 system was also equipped with a single LED (NSPW 300BS)
 125 having a maximum power output of 3 W. The LED can be driven
 126 in pulsed or continuous wave modes when incorporated with the
 127 light control circuit unit. The hardware setup of the developed
 128 optical system is shown in Fig. 2.

129 The light spectrum of the applied LED and the control circuit
 130 module are shown in Figs. 3(a) and 4, respectively. This arrange-
 131 ment enables a dual mode measurement capability in a single
 132 interferometer, where the continuous white light source can be
 133 applied for static surface profilometry while the stroboscopic

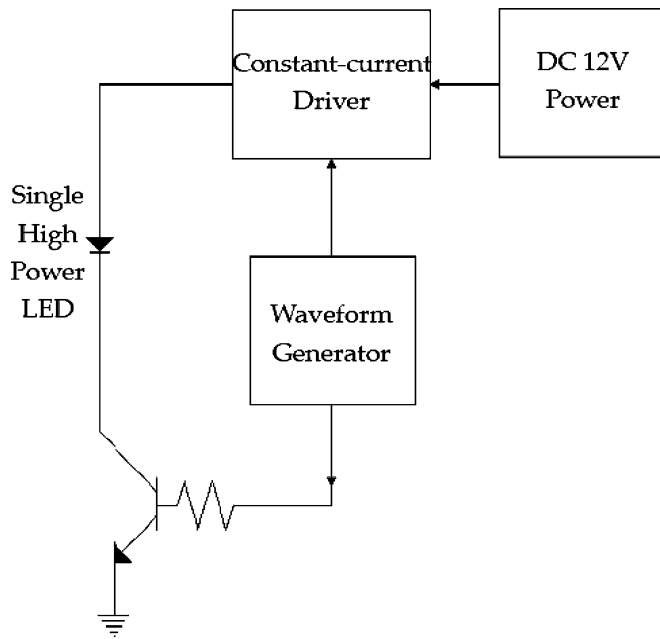


Fig. 4. Control circuit of the developed stroboscopic LED light module.

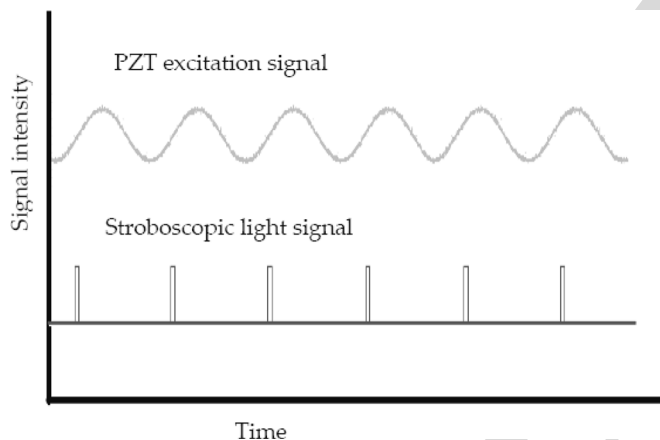


Fig. 5. Synchronized signals for stroboscopic signal and PZT vibration signal.

light source is utilized for dynamic vibratory measurements. The incident light from the light source module is collimated by a set of optical lenses for producing a parallel white light beam that illuminates the measured surface and reference mirror. A 5-V square signal with a duty cycle of less than 2% is employed to drive the control electronics of the stroboscopic LED. Meanwhile, the repeatability and timing accuracy of the LED firing electronics was evaluated by measuring the actual pulsed light with a photodetector (Thorlab PDA55 with a detection bandwidth of 10 MHz and sensible light spectrum ranging between 320 and 1100 nm). The time response of the light module can be seen in Fig. 3(b). The repeatability of the LED firing timing was identified to be within 2.1 ns for ± 1 standard deviation.

The MEMS testing sample is actuated by applying a sinusoidal voltage generated by a 20-MHz function waveform generator. The two driving signals mentioned earlier are accurately

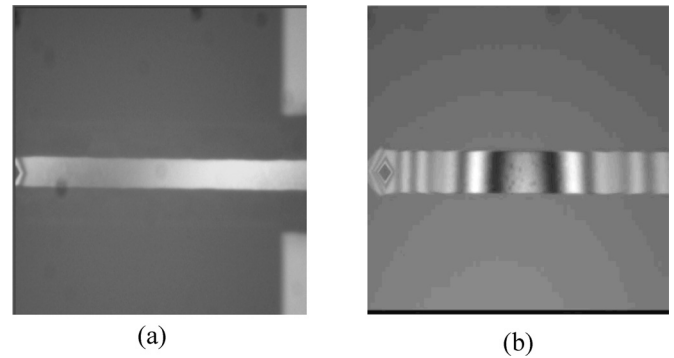


Fig. 6. AFM cantilever beam resonated at its secondary vibratory mode at a frequency of 1.067 MHz. (a) Blurred white light interferogram image when no strobed light was applied. (b) Good contrast interferogram image when a duty cycle of the strobed light set at 2% of the PZT driving signal was employed.

synchronized with an adjustable phase delay (shown in Fig. 5), in order to generate frozen interferograms. Although vibratory motion is damaging to the contrast quality of the conventional interferometric fringe images, this kind of stroboscopic source illumination is capable of capturing repeated images of the sample at the identical phase of oscillation such that the interference fringes can be unambiguously acquired. The whole system is mounted on a vibration-isolation optical table and placed in an environment with the minimum influence of external vibrations and disturbances. For the dynamic vibratory 3-D profilometry, the tested devices are rigidly secured on a silicon holder (*Pointprobe* silicon). The holder is mechanically fixed on a piezoelectric $\text{Pb}_x\text{ZrTiO}_{3(1-x)}$ (PZT) disk integrated with silver electrodes.

The measurement sensitivity and lateral resolution of the full-field out-of-plane vibratory motion of the developed system are primarily controlled by the diaphragm diameter and the global magnification of the optical system. With a 50-mm-diameter diaphragm and a $50\times$ Mirau objective, measurements of nanometric vibration amplitudes of MEMS with a spatial resolution equal to $1\ \mu\text{m}$ and a frequency bandwidth of more than 2 MHz can be achieved. Compared with previous ones, the developed system has the following two technical advances in stroboscopic interferometric measurement.

1) *Single light source for both static and dynamic out-of-plane measurement*: A single superluminescent LED of white light spectrum can be operated either in pulsed- or continuous-wave modes when incorporated with the homemade light-control electronics. With the developed light module, the conventional white light interferometric measurement systems can be transformed into a system capable of dynamic characterization.

2) *Generation of shorter duty cycle of the strobed electric signal*: The contrast quality of frozen interferometric fringes is mainly determined by the period of the duty cycle and the response bandwidth of the driving light circuit. Experimental results have demonstrated that less than 2% duty cycle of the stroboscopic LED pulse can achieve dynamic response bandwidth of up to 2 MHz.

An AFM cantilever measured by the developed device is shown in Fig. 6, in which the cantilever is vibrated at its

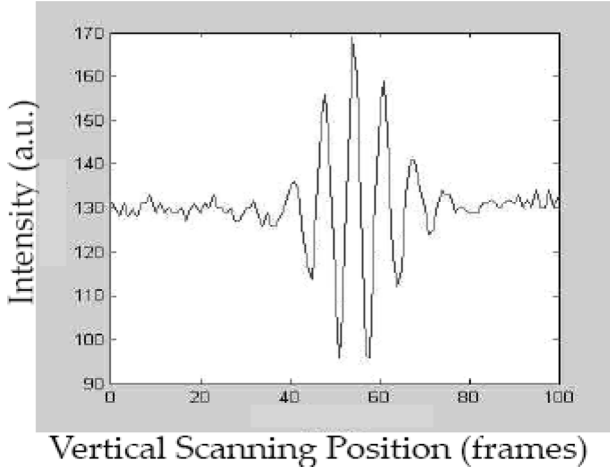


Fig. 7. White light scanning interferometric irradiance of static profile measurement of the Taylor Hobson standard step height.

191 secondary vibratory mode with a frequency of up to 1.067 MHz.
 192 Fig. 6(a) displays its white light interferogram image when no
 193 strobed light was applied. In contrast, Fig. 6(b) shows the corre-
 194 sponding interferogram image when a duty cycle of the strobed
 195 light set at 2% of the PZT driving signal was employed to freeze
 196 the interference fringes.

197 III. STROBOSCOPIC VIBRATION MEASUREMENTS ANALYSIS 198 AND INTERFEROGRAM PROCESSING

199 In white light interferometry, the position of the zero-order
 200 interference fringe obtained from the white light interferometry
 201 (WLI) is independent of the wavelength of light [8]. When the
 202 maximum fringe contrast is identified, no height ambiguities and
 203 no focus errors exist in the measurement of surface microstruc-
 204 ture. The detected intensity in the white light interferogram for
 205 the optical path difference ($2z$) between two optical arms can
 206 be expressed as follows [5], [9]:

$$I(x, y) = I_o[1 + C(z) \cos(4\pi z/\lambda_{mc} + \Delta\varphi)] \quad (1)$$

207 where

- 208 I_o background intensity;
- 209 λ_{mc} apparent mean source wavelength;
- 210 $\Delta\varphi$ local reflection phase shift difference;
- 211 $C(z)$ global contrast function.

212 An example of white light scanning interferometric irradiance
 213 obtained by measuring the Taylor Hobson standard step height
 214 is shown in Fig. 7. A model of the interference fringe $I(x, y)$
 215 can be further expressed by the following [9]:

$$\begin{aligned} g(z) &= \int_{k_l}^{k_u} \psi(k) \cos 2k(z - z_p) dk + C \\ &= m_c(z) \cos 2k_c z + m_s(z) \sin 2k_c z + C \end{aligned} \quad (2)$$

216 where

- 217 k angular wavenumber ($k = 2\pi/\lambda$);
- 218 z_p height of the surface of the object at point (x, y) ;

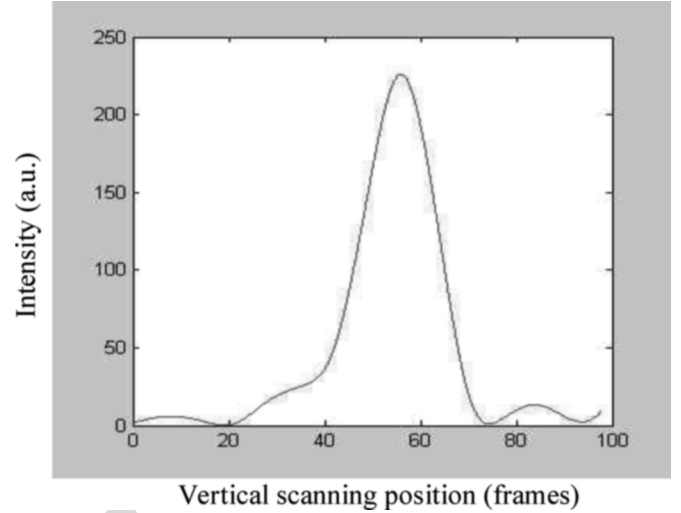


Fig. 8. Enveloped signal of the scanning interferometric irradiance shown in Fig. 7.

- $\psi(k)$ energy distribution of the incident beam to the CCD 219
- detector with respect to k ; 220
- k_c any fixed positive real number; 221
- 222

$$m_c(z) \text{ equals } \int_{k_l}^{k_u} \psi(k) \cos 2\{k(z - z_p) - k_c z\} dk;$$

$$m_s(z) \text{ equals } \int_{k_l}^{k_u} \psi(k) \sin 2\{k(z - z_p) - k_c z\} dk.$$

According to the definition of the interference fringe, the square 223
 envelope function is modeled as follows [9]: 224

$$\begin{aligned} r(z) &= \{m_c(z)\}^2 + \{m_s(z)\}^2 \\ &= \frac{2\Delta^2}{\pi^2} \left\{ \left(1 - \cos \frac{\pi z}{\Delta}\right) \left\{ \sum_{n=-\infty}^{\infty} \frac{f(z_{2n})}{z - z_{2n}} \right\}^2 \right. \\ &\quad \left. + \left(1 + \cos \frac{\pi z}{\Delta}\right) \left\{ \sum_{n=-\infty}^{\infty} \frac{f(z_{2n+1})}{z - z_{2n+1}} \right\}^2 \right\} \end{aligned} \quad (3)$$

where Δ is the sampling interval of the discrete interferometric 225
 fringes. 226

The square envelope function of the interference fringe obtained 227
 from WLI can be affected by some surface properties such as materials 228
 difference, surface roughness, and reflectivity. In addition, the inclined 229
 angle of the measured surface also affects the profile measurement since 230
 the reflected object beam cannot return back to the interferometer. Some of 231
 these challenges have been investigated in various cases [10], [11]. 232
 233

Using the aforementioned method, the static 3-D profile measurement 234
 can be reconstructed with the vertical resolution of up to 1 nm, and its 235
 repeatability within one standard deviation reaching a few nanometers. 236
 Figs. 8 and 9 illustrate the enveloped signal and the 3-D map reconstructed 237
 from the Taylor Hobson standard step height. The measurement accuracy has 238
 been demonstrated to be within 20 nm. The repeatability of static 239
 profilometry of the developed system was verified by measuring 240
 241

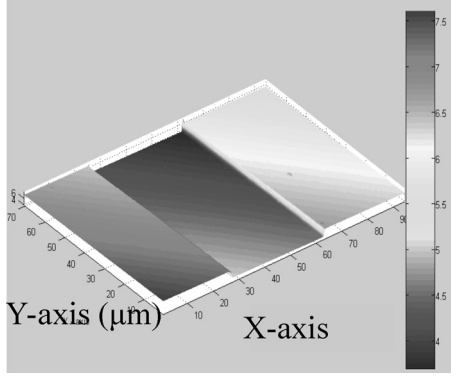


Fig. 9. 3-D profile measurement of the Taylor Hobson standard step height.

TABLE I
REPEATABILITY OF STATIC PROFILE MEASUREMENT USING
THE DEVELOPED SYSTEM

Repeatability of Static Profile Measurement			
Test standard piece	Taylor Hobson, Step Height Standard (3700.0 nm)		
Measurement standard	ISO 5436-1 Geometrical Product Specifications (GPS)		
Number of measurements	30		
Magnification of Mirau Interferometric Objective	10X	20X	50X
Measurement variation within $\pm\sigma$	20 nm	10 nm	4 nm

242 the step height (3700.0 ± 10.0 nm) of the Taylor Hobson masterpiece for 30 times. The obtained data were evaluated using
243 the International Standards Organization (ISO) 5436-1 Geometrical Product Specifications (GPS). Table I lists the measured
244 results for three kinds of Mirau interferometric objectives.

245 The earlier equation is suitable for the case when the continuous LED light source is applied. When the stroboscopic LED
246 light source is deployed, the contrast envelope has two peaks in its effective light power spectrum and the local vibration value
247 $z(x/G, y/G)$ of the tested sample at the resonance frequency can be described as [2]
248

$$z(x/G, y/G) = z_0(x/G, y/G) + a(x/G, y/G) \sin(\omega t + \phi_1((x/G, y/G))) \quad (4)$$

253 where

- 254 a vibration amplitude;
- 255 f vibration frequency;
- 256 ϕ_l phase lag between the driving signal and the device response;
- 257 G interferometric objective magnification.

258 For stroboscopic measurements, the sample is illuminated during a light pulsed time δT short with respect to the vibration
259 period $T = 1/f$. It can be demonstrated that if the duty cycle $\delta T/T$ is such that $\sin(n\delta T/T)/n\delta T/T \cong 1$ for all harmon-

TABLE II
DIMENSION SPECIFICATIONS OF THE TESTED AFM CANTILEVER BEAM

Technical Data	Typical Value	Typical Range	Specified Values
Thickness / μm	2	1.5 - 2.5	1.0 - 3.0
Mean Width / μm	50	45 - 55	42.5 - 57.5
Length / μm	450	445 - 455	440 - 460
Force Constant / (N/m)	0.2	0.07 - 0.4	0.02 - 0.77
Resonance Frequency / kHz	13	9 - 17	6 - 21

ics present in the signal, the intensity detected in stroboscopic measurements is given by the following:

$$I(x, y) = NT I_0 \left\{ 1 + C(z_0 + \Delta\phi + \alpha \sin(\omega t_0 + \phi_1)) \times \cos\left(\frac{4\pi}{\lambda_{mc}} z_0 + \Delta\phi + \frac{4\pi}{\lambda_{mc}} \alpha \sin(\omega t_0 + \phi_1)\right) \right\}. \quad (5)$$

265 Thus, when considering low vibration amplitudes and short light pulses such as $\sin(n\omega\delta T/2)/n\omega\delta T/2 \approx 1$, the detected intensity of stroboscopic interferograms being accumulated N times is given as

$$I(x, y) = N \int_{t_0 - \delta T/2}^{t_0 + \delta T/2} I(x, y, t) dt \cong N \delta T I_0 [1 + C(Z_0) \cos(B + \Delta\phi)]$$

where $B = \frac{4\pi}{\lambda_{mc}} z_0 + \frac{4\pi}{\lambda_{mc}} a \sin(\omega t_0 + \phi_1)$ (6)

269 The aforementioned intensity is similar to the one for the static measurement when the interferometric image is frozen using stroboscopic light. Fig. 6(b) illustrates the interferogram of AFM probe cantilever beams measured by the developed device in which the image contrast is as good as the one for its static mode.

IV. EXPERIMENTAL DESIGN AND RESULT ANALYSIS

A. Description of AFM Cantilever Microbeams

277 For illustration, the static and dynamic surface profiles of a contact-mode AFM cantilever microbeam were measured by the developed device. The microcantilever was fabricated by NANOSENSORS Corporation, and its detailed material specifications can be found in TABLE II.

282 Here, the material density (ρ), modulus of rigidity, Young's modulus, and yield strength of silicon were set at 2.33 g/cm^3 , $0.5 \times 10^{11} \text{ N/m}^2$, $1.69 \times 10^{11} \text{ N/m}^2$ (in the (1 1 0) direction), and $7 \times 10^9 \text{ N/m}^2$, respectively. Meanwhile, the cross section of the underlying cantilever is trapezoidal, in which the beam has two geometrical widths—a smaller one on the tip side and a broader one on the opposite side. Thus, the mean width represents the median between these two values.

B. Mechanical Analysis of AFM Microcantilever Beams

291 An AFM cantilever beam, in general, can be illustrated as Fig. 10, in which the important dimensions of the beam and

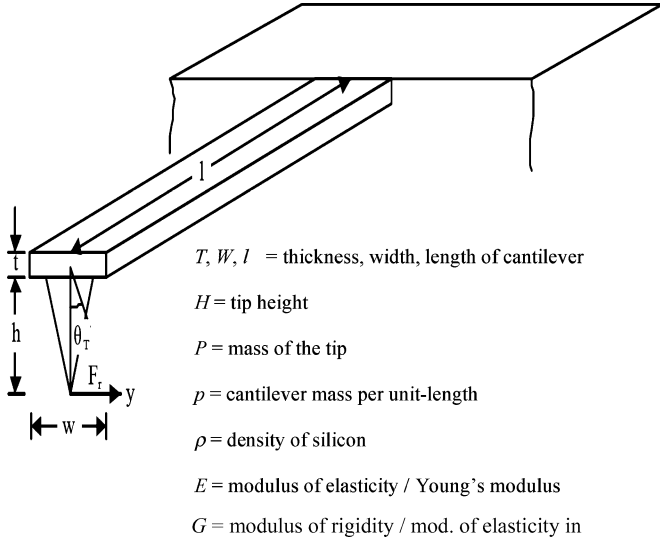


Fig. 10. Schematic diagram of an AFM cantilever beam.

293 probe tip are expressively depicted. By assuming that the beam
 294 is rigidly clamped at one end, the flexural resonant frequencies
 295 of a cantilever beam having a cross section A and an inertial
 296 moment I can be theoretically derived in (7) and [12]

$$f_n = \frac{\lambda_n^2}{2\pi} \frac{1}{L^2} \sqrt{\frac{EI}{\rho A}} \quad (7)$$

297 where

- 298 n mode order;
- 299 λ_n mode constant;
- 300 A cross section of cantilever beam;
- 301 L length of cantilever beam;
- 302 E effective Young's modulus;
- 303 I inertial moment;
- 304 ρ density of the beam material.

305 Equation (7) does not take the probe tip into consideration.
 306 When taking the tip mass into account and assuming the tip
 307 as a cone with height h and base diameter equal to h , the cor-
 308 rected flexural resonant frequencies of a cantilever beam can be
 309 modeled as (8) [13]

$$\begin{aligned} f_{\text{corr}} &= \frac{\sqrt{3}}{2\pi} \sqrt{\frac{EWT^3}{12(PL^3 + 0.236pL^4)}} \\ &= 0.276 \sqrt{\frac{EWT^3}{\rho(\pi h^3 L^3 + 0.283WTL^4)}}. \end{aligned} \quad (8)$$

310 For most cases, the crystallographic orientation along the can-
 311 tilver axis is aligned parallel to the (1 1 0) direction. According
 312 to the practical fracture limit used in the cantilever beam, the
 313 deflection of the cantilever at which the maximum stress in the
 314 lever equals 10% of the yield strength of the beam materials (1
 315 1 0) can be expressed as

$$Y = \frac{1}{15} \frac{\sum \lim L^2}{E T} \quad (9)$$

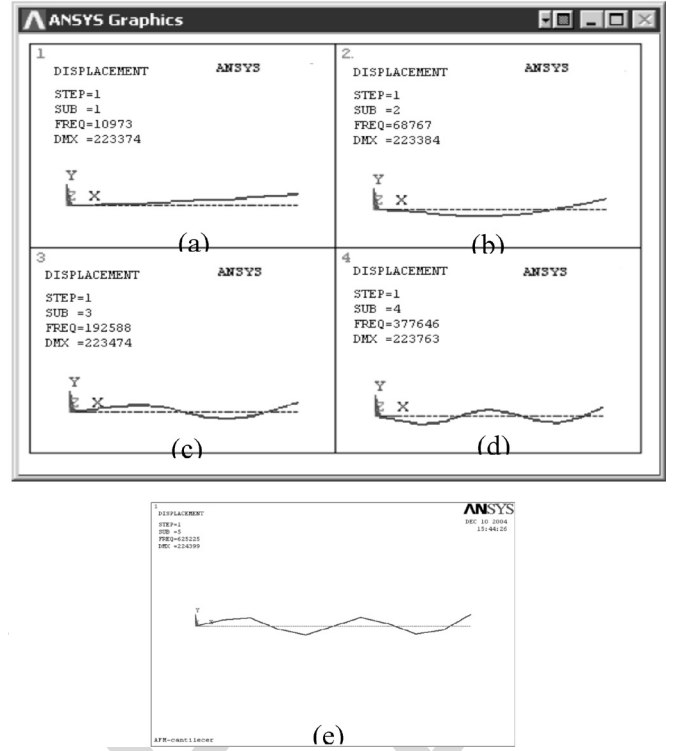


Fig. 11. ANSYS resonance analysis results of the cantilever obtained using ANSYS (the first five modes: (a), (b), (c), (d), and (e), respectively).

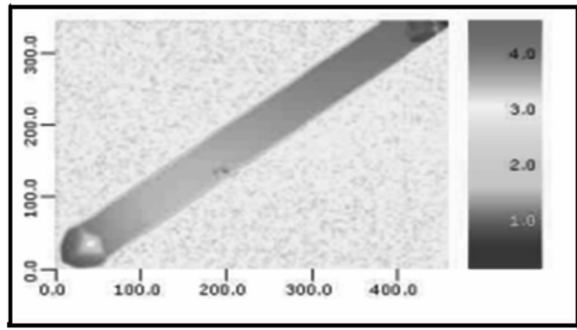
where Σ_{lim} is the yield strength of the beam material such as
 316 silicon. 317

C. Results and Analysis of the Theoretical Simulation 318

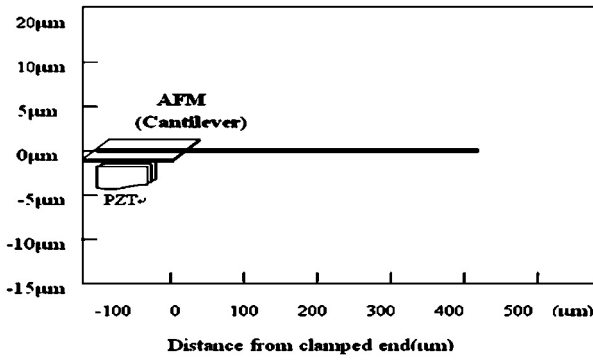
319 According to (8), a theoretical simulation on the beam dy-
 320 namic analysis for identifying its vibration modes was per-
 321 formed using ANSYS. The first five modes were simulated and
 322 shown in Fig. 11, in which its predicted natural frequencies were
 323 10.973, 68.767, 192.4, 377.5, and 623.9 kHz, respectively. To
 324 obtain the actual situation, the accurate mode frequencies were
 325 also measured by a laser Doppler interferometer and identified
 326 as 10.850, 68.60, 190.96, 387.0, and 643.2 kHz for the first five
 327 modes, respectively. It was noted that the predicted values were
 328 approximately consistent with the real values, and their differ-
 329 ences increased slightly when the vibratory mode was increased
 330 from 1% to 5%. Such errors in simulation can be attributed
 331 to the deviations in dimensions and material properties of the
 332 cantilever. 337

D. Measurement Results and Analysis of the Dynamic Vibratory 333 3-D Shape of the AFM Microcantilever Beam 334

335 Fig. 12 shows the static 3-D surface profile obtained by the
 336 developed white light interferometric scanning method. As can
 337 be seen, the maximum deflection of the cantilever beam was
 338 less than $1 \mu\text{m}$, which indicates that the beam was initially ori-
 339 entated at a relatively flat level. When performing the dynamic
 340 measurement of AFM cantilever beams, a $20\text{-}V_{\text{pp}}$ sinusoidal



(a)



(b)

Fig. 12. Static 3-D measurement results of the AFM cantilever beam (contact mode). (a) Static 3-D profile. (b) Schematic diagram of beam deflection.

341 voltage with the five vibration mode frequencies was applied to
 342 the PZT driver, and a 2% duty cycle was used for the strobo-
 343 scopic measurements. Stroboscopic measurements with white
 344 light vertical scanning interferometry of the vibration modes
 345 were performed at the first five frequency modes with a light
 346 pulse duty cycle of 2%. The LED and PZT driving signals were
 347 accurately synchronized with an adjustable phase delay.

348 Fig. 13 displays the first five resonance modes obtained using
 349 the stroboscopic measurement method developed. The measured
 350 mode shapes and cross section contours were obtained by
 351 profiling the vertical scanning contours along the microbeam
 352 length. The results were first corrected by the static deflection
 353 along the microbeam length. Theoretical mode shapes computed
 354 from the aforementioned ANSYS analysis were compared with
 355 the experimental ones and the two were convincingly consistent.

356 The slight inconsistency between the theoretical values and
 357 the measured ones may be attributed to the following two possible
 358 reasons.

- 359 1) *Dimension and material property deviation of the cantilever:* Potential measurement errors and the simplification used in the theoretical analysis may result in approximately 10% errors in the cantilever analysis.
- 360 361 2) *Inadequately clamped ends of the cantilever beam:* The cantilever beam is assumed to have ends perfectly clamped to its support base. Any inadequately clamped end may cause unexpected difference.

362 363 364 365 366 367 Dynamic measurement was also performed on a tapping-
 368 mode AFM cantilever beam (*Nanosensors NCLR*) with a different
 369 component specification. The second vibratory modes

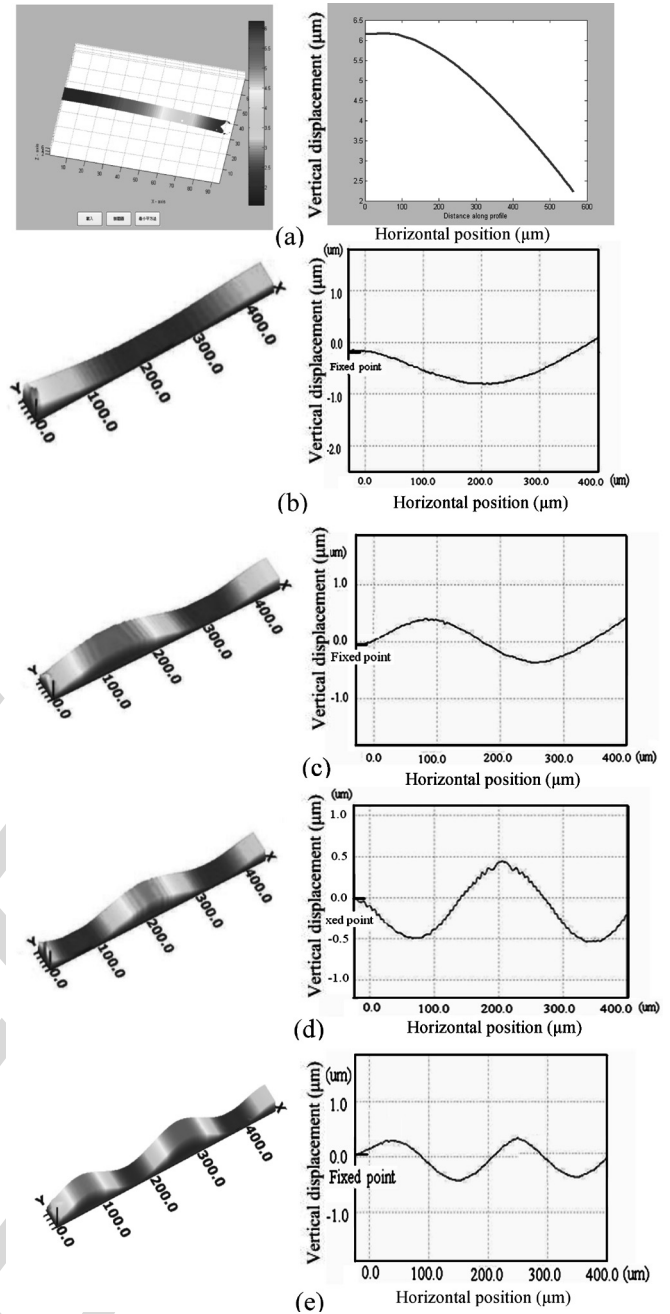


Fig. 13. 3-D map and cross section map of the first five resonance modes. (a)–(e) Sequentially represent the dynamic measurement results for the vibratory modes 1–5, respectively.

at 1192.1 kHz (the theoretical predicted value) was chosen
 370 to demonstrate the system capability in dynamic measurement
 371 at high resonant frequencies. By using a laser Doppler
 372 interferometer, the actual resonant frequency of this mode
 373 was identified as 1067 kHz. As shown in Fig. 14(a)–(h), the
 374 3-D maps sequentially represent the dynamic measurement
 375 results for successively different cycle times while Fig. 14(i)
 376 displays the cross section of the vibratory shape having the
 377 maximum vibratory amplitude of 400 nm. The maximum
 378 vibration amplitude was within 400 nm. The results have clearly
 379 demonstrated the capability of the developed optical system and
 380

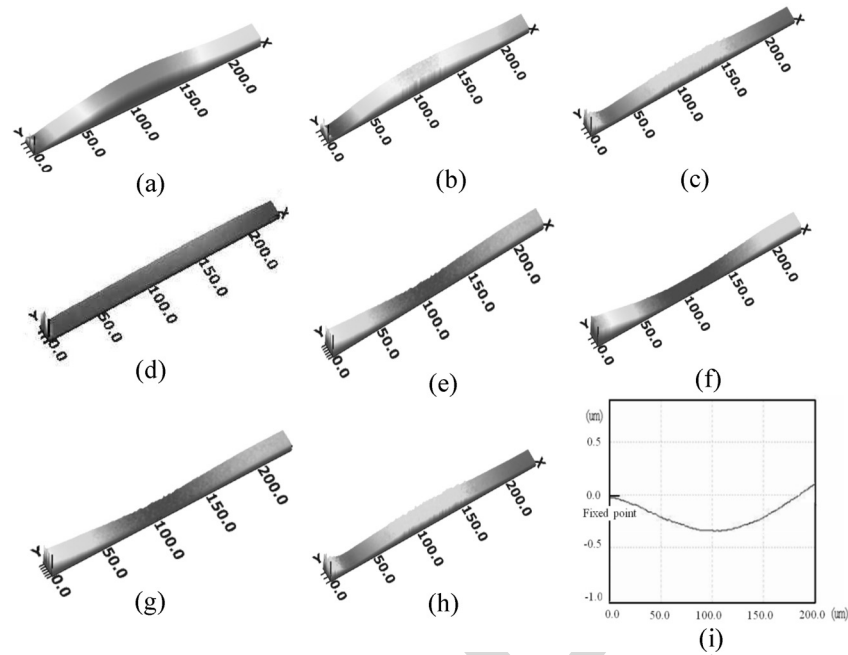


Fig. 14. Consequent 3-D vibratory maps of the second resonant mode (at frequency of 1.067 MHz). (a)–(h) Sequentially represent the dynamic measurement results for successively different cycle times, 12.5%, 25%, 37.5%, 50%, 62.5%, 75%, 87.5%, and 100%, respectively; and (i) displays the cross section of the vibratory shape having the maximum vibratory amplitude of 400 nm.

381 white light stroboscopic interferometry for the dynamic profile
382 measurement of complex vibratory behaviors being operated at
383 high frequency.

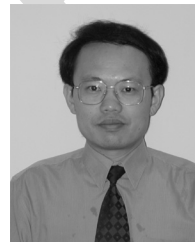
384 V. CONCLUSION

385 A dynamic surface profilometry involving white light inter-
386 ferometric scanning principle with a stroboscopic LED light
387 source was successfully developed for dynamic characteriza-
388 tion of AFM microcantilever beams. The experimental results
389 demonstrate that the developed method is suitable for accu-
390 rate full-field dynamic characterization of microdevices having
391 complex vibratory behaviors, and a large depth of field (DOF)
Q2 392 of hundredth micrometers can be achieved. The measured band-
393 width of the vibration mode shape can reach up to 1 MHz with a
394 depth detection resolution of 5 nm. Meanwhile, good agreement
395 between the theoretical simulated outcomes and experimental
396 results is found for 3-D vibratory characteristics of AFM mi-
397 crocantilever beams.

398 REFERENCES

- 399 [1] S. M. Spearing, "Materials issues in microelectromechanical systems
400 (MEMS)," *Acta Mater.*, vol. 48, pp. 179–196, 2000.
401 [2] A. Bosseboeuf and S. Petitgrand, "Characterization of the static and dy-
402 namic behaviour of M(O)EMS by optical techniques: Status and trends,"
403 *J. Micromech. Microeng.*, vol. 13, pp. S23–S33, 2003.
404 [3] K. E. Speller, H. Goldberg, J. Gannon, and E. Lawrence, "Unique MEMS
405 characterization solutions enabled by laser Doppler vibrometer measure-
406 ments," *Proc. SPIE*, vol. 4827, pp. 478–485, 2002.
407 [4] S. Ryder, K. B. Lee, X. F. Meng, and L. Lin, "AFM characterization of
408 out-of-plane high frequency microresonators," *Sens. Actuator A*, vol. 114,
409 pp. 135–140, 2004.
410 [5] K.-C. Fan, L.-C. Chen, C.-D. Lin, C. C. Chang, C.-F. Kuo, and J.-
411 T. Chou, "Development of dynamic 3-D surface profilometry using stro-
412 boscopic interferometric measurement and vertical scanning techniques,"
413 *J. Phys., Conf. Ser.*, vol. 13, pp. 51–54, 2005.

- [6] S. Bouwstra and B. Geijelaers, "On the resonant frequencies of micro-
414 bridges," in *Proc. Transducers*, 1991, vol. 1, pp. 538–542. 415
[7] M. R. Hart, R. A. Conant, K. Y. Lau, and R. S. Müller, "Stroboscopic
416 interferometer system for dynamic MEMS characterization," *J. Micro-*
417 *electromech. Syst.*, vol. 9, no. 4, pp. 409–418, 2000.
[8] J. C. Wyant, "White light interferometry," Opt. Sci. Center, Univ. Arizona,
418 Tucson, AZ. 419 Q3
[9] A. Hirabayashi, H. Ogawa, T. Mizutani, K. Nagai, and K. Kitagawa, "Fast
421 surface profiling by white-light interferometry by use of a new algorithm
422 based on sampling theory," *Appl. Opt.*, vol. 41, pp. 4876–4883, 2002. 423
[10] C. Quan *et al.*, "Study on the use of white light interferometry for
424 multifiber-end surface profile measurement," *Opt. Eng.*, vol. 45, no. 5,
425 p. 055603, 2006. 426
[11] C. Quan *et al.*, "Study on deformation of a microphone membrane using
427 multi-wavelength interferometry," *Opt. Commun.*, vol. 197, pp. 279–287,
428 2001. 429
[12] R. D. Blevins, *Formulas for Natural Frequency and Mode Shape*. Mel-
430 bourne, FL: Krieger, 1995, pp. 23–100. 431
[13] W. C. Young, *Roark's Formulas for Stress and Strain*. New York:
432 McGraw-Hill, 1989, pp. 1–154. 433



434 **Liang-Chia Chen** was born in Taiwan, R.O.C., 434
435 in 1963. He received the M.Sc.Eng. degree in me- 435
436 chanical engineering from National Taiwan Univer- 436
437 sity, Taipei, Taiwan, R.O.C., in 1989, and the Ph.D. 437
438 degree in advanced manufacturing and mechanical 438
439 engineering from the Centre for Manufacturing Re- 439
440 search (CAMR), University of South Australia, Ade- 440
441 laide, Australia, in 2000. 441

442 Since August 2001, he has been an Associate Pro- 442
443 fessor of automation technology at National Taipei 443
444 University of Technology, Taipei. He was a Research 444
445 Engineer with Gerard Industries of Australia for four years. His current research 445
446 interests include automatic optical inspection (AOI), nanomaterials fabrication 446
447 and control, reverse engineering for rapid manufacturing, and micro/nano 3-D 447
448 surface profilometry. He has published two book chapters and more than 50 448
449 international conference papers. He has refereed 45 journal papers. 449

450 Dr. Chen is a Graduate Member of the Institution of Engineers of Australia 450
451 and a member of the Society of Manufacturing Engineering. 452

453
454
455
456
457
458
459



Yao-Ting Huang received the M.S. degree in automation technology from National Taipei University of Technology, Taipei, Taiwan, R.O.C, in 2006.

His current research interests include image processing, automatic optical inspection (AOI), and nanometrology.

460
461
462
463
464
465
466
467
468
469
470
471
472
473
474
475
476
477
478
479



Kuang-Chao Fan was born in Taiwan, R.O.C., on January 4, 1950. He received the B.Sc. degree from National Taiwan University (NTU), Taipei, Taiwan, R.O.C., in 1972, the M.Sc. degree from the State University of New York at Buffalo, in 1976, and the Ph.D. degree from the University of Manchester Institute of Science and Technology, Manchester, U.K., in 1984, all in mechanical engineering.

He has been a Professor of mechanical engineering at NTU since August 1989. He was the Chairman of the Institute of Industrial Engineering, NTU, the

Chairman of Chinese Institute of Automation Technology, the Director of the Tjing Ling Industrial Research Institute, and the Associate Dean of the Engineering College, NTU. Since 2001, he has been the Yangtze Professor at Hefei University of Technology, Hefei, China. His current research interests include manufacturing metrology, precision machining, machine tool technology, micro/nanomeasurement, optical switches, and tactile sensors.

Dr. Fan was the Chairman of the Society of Manufacturing Engineers (SME) Taipei Chapter.

IEEE
Proof

- 481 Q1. Author: Please check the expansion of PI.
482 Q2. Author: Please explain "of hundredth micrometers" in Conclusion.
483 Q3. Author: Please provide the year for ref. [8].
484 Q4. Author: Please provide current affiliation of Y.-T. Huang.

IEEE
Proof